

MAKING MODERN LIVING POSSIBLE



## **Thermostatic expansion valves T 2 and TE 2**

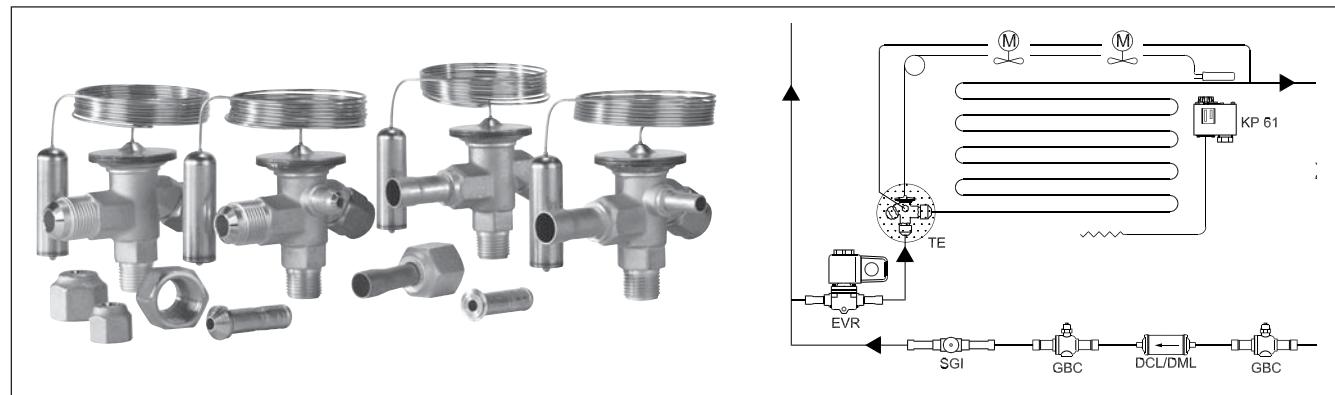
Technical brochure

Contents	Page
Introduction.....	3
Features.....	3
Thermostatic charges options .....	3
Regular valve options.....	3
Sizing .....	4, 5
Technical data .....	5
Ordering.....	6, 7
Extended capacity tables.....	8 - 14
R22.....	8, 9
R134a.....	10
R404A .....	11, 12
R507 .....	11, 12
Design.....	13
Dimensions and weights.....	13
T 2 / TE 2 operation and function.....	14
Identification .....	15

*Metric conversions*

1 psi = 0.07 bar  
 $\frac{5}{9}(t_1^{\circ}\text{F} - 32) = t_2^{\circ}\text{C}$   
1 ton = 3.5 kW  
1 in. = 25.4 mm  
1 ft = 0.3 m  
1 lb = 0.454 kg

## Introduction



Thermostatic expansion valves, type T 2 and TE 2 regulate the flow of refrigerant liquid into evaporators.

Injection is controlled by the refrigerant superheat.

T 2 valves are especially suitable for liquid injection into "dry" evaporators where the superheat at the evaporator outlet is proportional to the evaporator load.

The valves are available in rated capacities up to 4.5 TR (R22) and can be used in a wide range of applications:

- Conventional refrigeration systems
- Heat pump systems
- Air conditioning systems
- Specialty refrigeration appliances
- Liquid chillers
- Ice machines
- Transport refrigeration

This leaflet contains data and code numbers for our standard product program.

## Features

- Broad temperature range from -75 to 50°F:  
Applicable for deep freezing, refrigeration and air conditioning.
- Interchangeable orifice assembly  
Easier stocking, capacity matching, and service.
- Rated capacities from 0.15 to 4.5 TR (R22)
- Available with Maximum Operating Pressure (MOP)  
Used to protect the compressor motor against excessive evaporator pressure.
- *Stainless steel bulb*  
For easy self-aligning during installation and efficient heat transfer from pipe to bulb.

## Thermostatic charge options

Danfoss offers the following standard range of thermostatic charges:

Range N                    -40 to +50°F without MOP

Range N                    -40 to +50°F, MOP ~ 60°F

Range NM                 -40 to +25°F, MOP ~ 32°F

Range NL                 -40 to +5°F, MOP ~ 15°F

Range B<sup>1</sup>)              -75 to -15°F without MOP

Range B<sup>1</sup>)              -75 to -15°F, MOP ~ -4°F

<sup>1)</sup> T 2/TE 2 valves for range B are not available for R134a

## Regular valve options

*Static superheat (SS) (R22, R134a, and R404A):*

Valves without MOP: 9°F

Valves with MOP: 7°F

*Static superheat (SS) for R507:*

Valves without MOP: 11°F

Valves with MOP: 9°F

*Capillary tube length*  
5 ft

*Connection sizes*

Inlet x outlet: 3/8 in. flare x 1/2 in. flare

3/8 in. flare x 1/2 in. ODF

3/8 in. ODF\*) x 1/2 in. flare

3/8 in. ODF\*) x 1/2 in. ODF

1/4 in. ODF\*) x 1/2 in. flare

1/4 in. ODF\*) x 1/2 in. ODF

\*) Requires a solder adapter

*External equalization:*

1/4 in. flare (for all flare outlet bodies)

1/4 in. ODF (for all ODF outlet bodies)

**Sizing**

For optimum performance, it is important to select a T 2 or TE 2 valve according to system conditions and application. Selecting an incorrect valve will result in operational difficulties or poor system performance. The following procedure will help you select the correct valve for your needs.

**Example:**

Refrigerant = R22

Evaporator capacity:  
 $Q_e = 13,000 \text{ Btu (1.08 TR)}$

Evaporating temperature:  
 $t_e = +40^\circ\text{F}$  (equals  $p_e = 70 \text{ psig}$ )

Condensing temperature:  
 $t_c = +100^\circ\text{F}$  (equals  $p_c = 195 \text{ psig}$ )

Liquid refrigerant temperature:  
 $t_l = +80^\circ\text{F}$

**Step 1**

*Determine the pressure drop across the valve.*  
The pressure drop,  $\Delta p$ , is calculated by the formula:

$$\Delta p = p_c - p_e - pdw$$

where

$p_c$  = condensing pressure

$p_e$  = evaporating pressure, and

$pdw$  = the sum of other pressure drops in the liquid line, evaporator and distributor.

The pressures  $p_c$  and  $p_e$  can be found by using the design condensing and evaporating temperatures at the saturated vapor point, then using a pressure-temperature chart or a Danfoss refrigerant slide to convert the temperatures to pressures.

In this example, the pressure drop across the valve will be:

$$\Delta p = p_c - p_e - pdw = 195 - 70 - 0 = 125 \text{ psi.}$$

**Step 2**

*Determine required valve capacity.*

Use the design evaporator capacity,  $Q_e$ , to select the required valve size at a given evaporating temperature. If necessary, correct the evaporator capacity for subcooling. Subcooled liquid refrigerant entering the evaporator increases evaporator capacity, so that a smaller valve may be required.

In this example, the subcooling is:

$$\Delta t_{sub} = t_c - t_l = 100 - 80 = 20^\circ\text{F}$$

From the subcooling correction factor table on page 9, we find the appropriate correction factor  $F_{sc}$  equals 0.94 for  $\Delta t_{sub} = 20^\circ\text{F}$ . Now determine the required valve capacity by multiplying the evaporator capacity by the correction factor for subcooling.

**Required valve capacity**

$$Q_e \times F_{sc} = 1.08 \times 0.94 = 1.02 \text{ TR}$$

**Step 3**

Use the calculated valve capacity to select the corresponding orifice size from the capacity table for R22 as indicated below.

Note that the expansion valve capacity must be equal to or slightly more than the corrected evaporator capacity.

In this sizing example, orifice 02 will be suitable.

Capacities (R22)		Capacity in tons, range N: -40 to +50°F								
Valve type	Orifice no.	Pressure drop across valve $\Delta p$ psi								
		40	60	80	100	125	150	175	200	
<b>Evaporating temperature + 40°F</b>										
TX 2/ TEX 2	0X	0.12	0.14	0.15	0.16	0.17	0.18	0.18	0.19	
	00	0.27	0.31	0.34	0.36	0.38	0.39	0.40	0.40	
	01	0.65	0.75	0.82	0.87	0.92	0.96	0.98	1.00	
	02	0.91	1.06	1.18	1.27	<b>1.36</b>	1.43	1.47	1.50	
	03	1.62	1.90	2.11	2.27	2.43	2.54	2.62	2.67	
	04	2.42	2.84	3.15	3.39	3.63	3.80	3.92	3.99	
	05	3.06	3.59	3.98	4.29	4.58	4.79	4.94	5.03	
	06	3.76	4.40	4.87	5.24	5.59	5.84	6.02	6.13	

**Metric conversions**

1 psi = 0.07 bar

$\frac{5}{9}(t_1^\circ\text{F} - 32) = t_2^\circ\text{C}$

1 ton = 3.5 kW

**Sizing (continued)**
**Step 4**

*Select a thermostatic charge.*  
Danfoss offers a universal wide range thermostatic charge.

Range N is suited for most applications in the normal temperature range.

Ranges NM, NL and B are available for special low temperature applications.

**Step 5**

Determine if external equalization is required.

Note! External equalization is always required if a distributor is used or if there is an appreciable difference in pressure from the valve outlet to the bulb location.

Finally, determine connection sizes, then find the valve's code number from the tables on page 7.

**Technical data**

*Maximum bulb temperature*  
212°F

*Static superheat (SS) (R22, R134a, and R404A):*

Valves without MOP: 9°F

Valves with MOP: 7°F

*Maximum valve body temperature*  
250°F

*Static superheat (SS) for R507:*

Valves without MOP: 11°F

Valves with MOP: 9°F

*Maximum working pressure*  
MWP = 400 psig

*Standard refrigerants*

R22, R134a and R404A/R507

*Maximum test pressure*  
p' = 520 psig

T 2 and TE 2 valves are continually evaluated for use with newer refrigerants.

For further information, please contact Danfoss.

**MOP-points**

Refrigerant	Range N -40 → +50°F	Range NM -40 → +25°F	Range NL -40 → +5°F	Range B -75 → -15°F
	MOP point for evaporating temperature t <sub>e</sub> and evaporating pressure p <sub>e</sub> <sup>1)</sup>			
	t <sub>e</sub> = +60°F	t <sub>e</sub> = +32°F	t <sub>e</sub> = +15°F	t <sub>e</sub> = -4°F
R22	p <sub>e</sub> = 100 psig	p <sub>e</sub> = 60 psig	p <sub>e</sub> = 35 psig	p <sub>e</sub> = 20 psig
R134a	p <sub>e</sub> = 55 psig	p <sub>e</sub> = 30 psig	p <sub>e</sub> = 15 psig	
R404A/R507	p <sub>e</sub> = 120 psig	p <sub>e</sub> = 75 psig	p <sub>e</sub> = 50 psig	p <sub>e</sub> = 30 psig

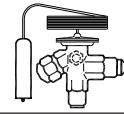
**MOP valves**

To avoid charge migration when using MOP valves, the bulb temperature must always be lower than the temperature of the thermostatic element.

**Metric conversions**

1 psi = 0.07 bar

5/9 (t<sub>1</sub>°F - 32) = t<sub>2</sub>°C

**Ordering, Components with flare × flare connections**
*Thermostatic element and valve body with bulb strap (without orifice and filter)*


Refrigerant	Valve type	Pressure equalization 1)	Capillary tube ft	Connection Inlet x outlet in. x in.	Code no.					
					Range N -40 to +50°F		Range NM -40 to +25°F		Range NL -40 to +5°F	
					Without MOP	With MOP	With MOP	With MOP	Without MOP	With MOP
R22	TX 2	Int.	5	3/8 × 1/2	068Z3206	068Z3208	068Z3224	068Z3226	068Z3207	068Z3228
	TEX 2	Ext.	5	3/8 × 1/2	068Z3209	068Z3211	068Z3225	068Z3227	068Z3210	068Z3229
R134a	TN 2	Int.	5	3/8 × 1/2	068Z3346	068Z3347	068Z3393	068Z3369		
	TEN 2	Ext.	5	3/8 × 1/2	068Z3348	068Z3349	068Z3392	068Z3370		
R404A/ R507	TS 2	Int.	5	3/8 × 1/2	068Z3400	068Z3402	068Z3406	068Z3408	068Z3401	068Z3410
	TES 2	Ext.	5	3/8 × 1/2	068Z3403	068Z3405	068Z3407	068Z3409	068Z3404	068Z3411

<sup>1)</sup> Externally equalized connections are 1/4 in. flare.

**Orifice assembly with filter**


Rated capacity is based on:  
 Evaporating temperature  
 $t_e = +40^\circ\text{F}$  for range N and  
 $t_e = -20^\circ\text{F}$  for range B  
 Condensing temperature  
 $t_c = +90^\circ\text{F}$   
 Refrigerant liquid temperature ahead of valve  $t_l = +80^\circ\text{F}$

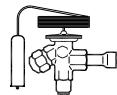
**Rated capacity in tons (TR)**

Orifice no.	Range N: -40 to +50°F			Range B: -75 to -15°F		Code no.
	R22	R134a	R404A/R507	R22	R404A/R507	
0X	0.15	0.11	0.11	0.15	0.11	068-2002
00	0.3	0.25	0.21	0.2	0.21	068-2003
01	0.7	0.5	0.45	0.3	0.45	068-2010
02	1.0	0.8	0.6	0.6	0.6	068-2015
03	1.5	1.3	1.2	0.8	1.0	068-2006
04	2.3	1.9	1.7	1.2	1.4	068-2007
05	3.0	2.5	2.2	1.5	1.7	068-2008
06	4.5	3.0	2.6	2.0	1.9	068-2009

**Metric conversions**

1 psi = 0.07 bar  
 $\frac{5}{9} (t_1^\circ\text{F} - 32) = t_2^\circ\text{C}$   
 1 ton = 3.5 kW  
 1 in. = 25.4 mm  
 1 ft = 0.3 m

## Ordering, components with flare × solder connections



Thermostatic element and valve body with bulb strap (without orifice and filter)

Refrigerant	Valve type	Pressure equalization <sup>1)</sup>	Capillary tube ft	Connection		Code no.					
				Inlet flare in.	Outlet ODF solder in.	Range N – 40 to + 50°F		Range NL – 40 to + 5°F	Range B – 75 to – 15°F		
						without MOP	with MOP		with MOP	without MOP	with MOP
R22	TX 2	Int.	5	3/8	1/2	068Z3281	068Z3287			068Z3357	068Z3319
	TEX 2	Ext.	5	3/8	1/2	068Z3284	068Z3290			068Z3359	068Z3320
R134a	TN 2	Int.	5	3/8	1/2	068Z3383					
	TEN 2	Ext.	5	3/8	1/2	068Z3385	068Z3389				
R404A/R507	TS 2	Int.	5	3/8	1/2	068Z3414	068Z3416			068Z3418	068Z3420
	TES 2	Ext.	5	3/8	1/2	068Z3415	068Z3417	068Z3430	068Z3419	068Z3421	

<sup>1)</sup> Externally equalized connections are 1/4 in. ODF solder.

## Orifice assembly with filter



Rated capacity is based on:

Evaporating temperature

 $t_e = + 40^\circ\text{F}$  for range N and $t_e = - 20^\circ\text{F}$  for range B

Condensing temperature

 $t_c = + 90^\circ\text{F}$ Refrigerant liquid temperature ahead of valve  $t_l = + 80^\circ\text{F}$ 

## Rated capacity in tons (TR)

Orifice no. <sup>1)</sup>	Range N: – 40 to + 50°F			Range B: – 75 to – 15°F		Code no.
	R22	R134a	R404A/R507	R22	R404A/R507	
OX	0.15	0.11	0.11	0.15	0.11	068-2002
00	0.3	0.25	0.21	0.2	0.21	068-2003
01	0.7	0.5	0.45	0.3	0.45	068-2010
02	1.0	0.8	0.6	0.6	0.6	068-2015
03	1.5	1.3	1.2	0.8	1.0	068-2006
04	2.3	1.9	1.7	1.2	1.4	068-2007
05	3.0	2.5	2.2	1.5	1.7	068-2008
06	4.5	3.0	2.6	2.0	1.9	068-2009

<sup>1)</sup> These orifice assemblies cannot be used together with solder adapters. Please see adapter information below.

## Solder adapter



The adapter is for use with thermostatic expansion valves type T 2 and TE 2 with flare × solder connections. When the adapter is correctly fitted it meets DIN 8964 sealing requirements.

The adapter offers the following advantages without unsoldering the valve from the system:

- The orifice assembly can be replaced.
- The filter can be cleaned or replaced.

When using the solder adapter, a special orifice assembly is required. Please use the following tables to select both the appropriate adapter and orifice assembly.

## Solder adapter without orifice assembly and filter

Connection ODF solder	Code no.
1/4 in.	068-2062
3/8 in.	068-2060

## Filter for solder adapter

Description	Code no.
Filter excl. orifice assembly	068-0015

## Orifice assembly and filter for use with solder adapter

Orifice no.	Code no.
0X	068-2089
00	068-2090
01	068-2091
02	068-2092
03	068-2093
04	068-2094
05	068-2095
06	068-2096

## Metric conversions

1 psi = 0.07 bar

 $5/9 (t_1^\circ\text{F} - 32) = t_2^\circ\text{C}$ 

1 ton = 3.5 kW

1 in. = 25.4 mm

1 ft = 0.3 m

**Capacities (TR)**

Range N: -40°F to +50°F

Valve type	Orifice no.	Pressure drop across valve Δp psi							
		40	60	80	100	125	150	175	200

**Evaporating temperature + 50°F**

TX 2/ TEX 2	0X	0.12	0.14	0.15	0.16	0.17	0.18	0.19	0.19
	00	0.28	0.32	0.34	0.37	0.39	0.40	0.41	0.41
	01	0.70	0.81	0.88	0.94	1.00	1.03	1.06	1.07
	02	0.98	1.16	1.29	1.39	1.49	1.56	1.61	1.64
	03	1.75	2.06	2.30	2.48	2.65	2.77	2.86	2.91
	04	2.63	3.10	3.45	3.72	3.97	4.16	4.28	4.36
	05	3.33	3.92	4.35	4.69	5.00	5.23	5.39	5.48
	06	4.09	4.8	5.33	5.73	6.11	6.39	6.57	6.68

**Evaporating temperature + 40°F**

TX 2/ TEX 2	0X	0.12	0.14	0.15	0.16	0.17	0.18	0.18	0.19
	00	0.27	0.31	0.34	0.36	0.38	0.39	0.40	0.40
	01	0.65	0.75	0.82	0.87	0.92	0.96	0.98	1.00
	02	0.91	1.06	1.18	1.27	1.36	1.43	1.47	1.50
	03	1.62	1.90	2.11	2.27	2.43	2.54	2.62	2.67
	04	2.42	2.84	3.15	3.39	3.63	3.80	3.92	3.99
	05	3.06	3.59	3.98	4.29	4.58	4.79	4.94	5.03
	06	3.76	4.40	4.87	5.24	5.59	5.84	6.02	6.13

**Evaporating temperature + 20°F**

TX 2/ TEX 2	0X	0.12	0.13	0.15	0.16	0.17	0.17	0.18	0.18
	00	0.25	0.29	0.31	0.33	0.35	0.36	0.37	0.38
	01	0.54	0.62	0.68	0.72	0.77	0.8	0.83	0.84
	02	0.75	0.88	0.97	1.05	1.12	1.17	1.21	1.23
	03	1.35	1.57	1.74	1.87	1.99	2.09	2.15	2.20
	04	2.00	2.33	2.58	2.77	2.95	3.10	3.20	3.26
	05	2.54	2.96	3.26	3.50	3.74	3.91	4.04	4.12
	06	3.11	3.62	3.99	4.28	4.56	4.78	4.93	5.03

*Correction for subcooling Δt<sub>sub</sub>*

When subcooling Δt<sub>sub</sub> ahead of the expansion valve is other than 10°F, adjust the evaporator capacity by multiplying by the appropriate correction factor found in the following table.

Refrigerant	0°F	10°F	20°F	30°F	40°F	50°F	60°F	70°F	80°F	90°F
	R22	1.08	1.00	0.94	0.89	0.85	0.81	0.78	0.74	0.68

*Metric conversions*

1 psi = 0.07 bar

 $\frac{5}{9}(t_1 - 32) = t_2$  °C

1 ton = 3.5 kW

**Capacities (TR) (continued)**
*Range B: -75°F to -15°F*
**R22**

Valve type	Orifice no.	Pressure drop across valve $\Delta p$ psi						
		40	60	80	100	125	150	175

**Evaporating temperature – 15°F**

TX 2/ TEX 2	00	0.21	0.24	0.26	0.27	0.29	0.31	0.32	0.32
	01	0.37	0.42	0.46	0.49	0.52	0.55	0.57	0.58
	02	0.52	0.60	0.65	0.70	0.74	0.77	0.80	0.81
	03	0.93	1.07	1.17	1.25	1.33	1.39	1.43	1.46
	04	1.37	1.57	1.72	1.84	1.95	2.04	2.11	2.16
	05	1.74	2.00	2.18	2.33	2.48	2.59	2.68	2.74
	06	2.13	2.44	2.66	2.84	3.02	3.16	3.27	3.35

**Evaporating temperature – 30°F**

TX 2/ TEX 2	00	0.19	0.22	0.23	0.25	0.27	0.28	0.29	0.29
	01	0.32	0.36	0.39	0.41	0.44	0.46	0.48	0.49
	02	0.44	0.5	0.55	0.58	0.61	0.64	0.66	0.67
	03	0.79	0.9	0.98	1.04	1.1	1.15	1.19	1.21
	04	1.15	1.31	1.43	1.52	1.62	1.69	1.75	1.79
	05	1.46	1.67	1.82	1.94	2.05	2.15	2.22	2.27
	06	1.78	2.04	2.22	2.36	2.51	2.62	2.72	2.78

**Evaporating temperature – 40°F**

TX 2/ TEX 2	00	0.18	0.2	0.22	0.24	0.25	0.26	0.27	0.28
	01	0.28	0.32	0.35	0.37	0.39	0.41	0.42	0.43
	02	0.39	0.45	0.48	0.51	0.54	0.57	0.58	0.59
	03	0.70	0.8	0.87	0.92	0.97	1.02	1.05	1.07
	04	1.02	1.16	1.27	1.35	1.43	1.49	1.55	1.58
	05	1.30	1.48	1.61	1.71	1.82	1.90	1.97	2.01
	06	1.59	1.81	1.97	2.09	2.22	2.32	2.40	2.46

**Correction for subcooling  $\Delta t_{sub}$** 

When subcooling  $\Delta t_{sub}$  ahead of the expansion valve is other than 10°F, adjust the evaporator capacity by multiplying by the appropriate correction factor found in the following table.

Refrigerant	0°F	10°F	20°F	30°F	40°F	50°F	60°F	70°F	80°F	90°F
	R22	1.08	1.00	0.94	0.89	0.85	0.81	0.78	0.74	0.71

**Metric conversions**

1 psi = 0.07 bar

 $\frac{5}{9}(t_1 - 32) = t_2$  °C

1 ton = 3.5 kW

## Capacities (TR) (continued)

Range N: -40 → +50°F

R134a

Type	Orifice no.	Pressure drop across valve Δp psi				
		40	60	80	100	125
<b>Evaporating temperature + 50°F</b>						
TN 2/ TEN 2	0X	0.10	0.12	0.12	0.13	0.13
	00	0.22	0.25	0.26	0.27	0.28
	01	0.47	0.53	0.57	0.60	0.62
	02	0.66	0.76	0.82	0.87	0.90
	03	1.17	1.35	1.47	1.55	1.61
	04	1.74	2.01	2.18	2.30	2.39
	05	2.21	2.55	2.77	2.91	3.03
TN 2/ TEN 2	06	2.71	3.12	3.38	3.56	3.69
<b>Evaporating temperature + 40°F</b>						
0X	0.10	0.11	0.12	0.13	0.13	
00	0.21	0.23	0.25	0.26	0.27	
01	0.43	0.48	0.52	0.54	0.56	
02	0.59	0.68	0.74	0.78	0.81	
03	1.06	1.22	1.33	1.40	1.45	
TN 2/ TEN 2	04	1.57	1.8	1.96	2.07	2.15
	05	1.99	2.29	2.49	2.62	2.72
	06	2.44	2.80	3.04	3.20	3.32
<b>Evaporating temperature + 20°F</b>						
TN 2/ TEN 2	0X	0.09	0.11	0.11	0.12	0.12
	00	0.19	0.21	0.22	0.23	0.24
	01	0.34	0.39	0.42	0.44	0.45
	02	0.47	0.54	0.59	0.62	0.64
	03	0.85	0.97	1.05	1.11	1.15
	04	1.25	1.42	1.54	1.62	1.69
	06	1.94	2.21	2.39	2.52	2.64
<b>Evaporating temperature - 20°F</b>						
TN 2/ TEN 2	0X	0.08	0.09	0.09	0.10	0.1
	00	0.15	0.16	0.17	0.18	0.18
	01	0.21	0.23	0.25	0.26	0.27
	02	0.29	0.32	0.35	0.36	0.37
	03	0.52	0.58	0.62	0.65	0.67
	04	0.75	0.84	0.90	0.95	0.98
	06	1.16	1.31	1.41	1.48	1.53
<b>Evaporating temperature - 40°F</b>						
TN 2/ TEN 2	0X	0.07	0.08	0.08	0.09	0.09
	00	0.13	0.14	0.15	0.16	0.16
	01	0.17	0.19	0.20	0.21	0.22
	02	0.23	0.26	0.27	0.29	0.29
	03	0.41	0.46	0.49	0.51	0.53
	04	0.59	0.66	0.71	0.74	0.77
	06	0.76	0.84	0.91	0.95	0.98

Correction for subcooling  $\Delta t_{sub}$ 

When subcooling  $\Delta t_{sub}$  ahead of the expansion valve is other than 10°F, adjust the evaporator capacity by multiplying by the appropriate correction factor found in the following table.

Refrigerant	0°F	10°F	20°F	30°F	40°F	50°F	60°F	70°F	80°F	90°F
R134a	1.10	1.00	0.93	0.88	0.82	0.78	0.74	0.70	0.67	0.64

## Metric conversions

1 psi = 0.07 bar

 $\frac{5}{9}(t_1 - 32) = t_2$  °C

1 ton = 3.5 kW

**Capacities (TR) (continued)**
*Range N: -40°F to +50°F*

Valve type	Orifice no.	Pressure drop across valve $\Delta p$ psi							
		40	60	80	100	125	150	175	200

**Evaporating temperature + 50°F**

TS 2/ TES 2	0X	0.09	0.11	0.11	0.12	0.12	0.12	0.12	0.12
	00	0.20	0.22	0.23	0.24	0.25	0.25	0.26	0.26
	01	0.43	0.48	0.52	0.54	0.57	0.57	0.58	0.58
	02	0.59	0.69	0.75	0.80	0.83	0.85	0.86	0.87
	03	1.05	1.23	1.35	1.42	1.49	1.52	1.54	1.56
	04	1.56	1.83	2.00	2.11	2.20	2.27	2.29	2.31
	05	1.99	2.31	2.53	2.66	2.78	2.86	2.89	2.91
	06	2.43	2.84	3.10	3.25	3.39	3.49	3.52	3.54

**Evaporating temperature + 40°F**

TS 2/ TES 2	0X	0.08	0.11	0.11	0.12	0.12	0.12	0.12	0.12
	00	0.19	0.21	0.22	0.24	0.24	0.24	0.24	0.25
	01	0.39	0.45	0.48	0.50	0.52	0.53	0.53	0.53
	02	0.54	0.63	0.70	0.73	0.76	0.77	0.78	0.78
	03	0.97	1.12	1.24	1.29	1.36	1.40	1.40	1.41
	04	1.43	1.66	1.83	1.93	2.01	2.06	2.07	2.08
	05	1.82	2.11	2.31	2.43	2.54	2.61	2.62	2.63
	06	2.23	2.59	2.84	2.97	3.10	3.17	3.19	3.20

**Evaporating temperature + 20°F**

TS 2/ TES 2	0X	0.09	0.10	0.11	0.11	0.11	0.11	0.11	0.11
	00	0.17	0.20	0.21	0.22	0.22	0.23	0.23	0.22
	01	0.33	0.37	0.41	0.42	0.44	0.45	0.45	0.44
	02	0.45	0.52	0.58	0.60	0.63	0.64	0.64	0.63
	03	0.81	0.94	1.02	1.08	1.12	1.14	1.15	1.13
	04	1.18	1.38	1.51	1.59	1.66	1.70	1.70	1.68
	05	1.50	1.75	1.91	2.01	2.09	2.15	2.15	2.13
	06	1.84	2.13	2.34	2.46	2.56	2.62	2.62	2.60

<sup>1)</sup> The capacities for R507 deviate less than  $\pm 2\%$  from the capacities for R404A.

*Correction for subcooling  $\Delta t_{sub}$* 

When subcooling  $\Delta t_{sub}$  ahead of the expansion valve is other than 10°F, adjust the evaporator capacity by multiplying by the appropriate correction factor found in the following table.

Refrigerant	0°F	10°F	20°F	30°F	40°F	50°F	60°F	70°F	80°F	90°F
R404A/R507	1.18	1.00	0.89	0.81	0.74	0.68	0.63	0.60	0.56	0.53

*Metric conversions*

1 psi = 0.07 bar

 $5/9 (t_1^{\circ}\text{F} - 32) = t_2^{\circ}\text{C}$ 

1 ton = 3.5 kW

**Capacities (TR) (continued)**

Range B: -75°F to -15°F

**R404A and R507<sup>1)</sup>**

Valve type	Orifice no.	Pressure drop across valve Δp psi							
		40	60	80	100	125	150	175	200

**Evaporating temperature - 15°F**

TS 2/ TES 2	00	0.17	0.19	0.20	0.20	0.21	0.21	0.21	0.21
	01	0.29	0.34	0.36	0.37	0.37	0.38	0.38	0.37
	02	0.40	0.47	0.49	0.53	0.53	0.54	0.53	0.53
	03	0.72	0.84	0.89	0.93	0.96	0.96	0.96	0.95
	04	1.06	1.22	1.31	1.38	1.40	1.42	1.42	1.40
	05	1.35	1.56	1.67	1.74	1.79	1.80	1.80	1.79
	06	1.66	1.91	2.04	2.13	2.18	2.19	2.19	2.18

**Evaporating temperature - 30°F**

TS 2/ TES 2	00	0.16	0.17	0.18	0.19	0.19	0.19	0.18	0.19
	01	0.24	0.28	0.29	0.30	0.31	0.31	0.32	0.31
	02	0.34	0.38	0.41	0.43	0.44	0.43	0.43	0.42
	03	0.60	0.69	0.74	0.77	0.78	0.78	0.78	0.77
	04	0.88	1.01	1.07	1.12	1.14	1.15	1.14	1.13
	05	1.12	1.27	1.37	1.43	1.46	1.46	1.46	1.44
	06	1.38	1.56	1.67	1.74	1.78	1.79	1.79	1.77

**Evaporating temperature - 40°F**

TS 2/ TES 2	00	0.12	0.14	0.15	0.15	0.15	0.16	0.16	0.16
	01	0.16	0.18	0.20	0.21	0.21	0.23	0.22	0.22
	02	0.21	0.24	0.27	0.28	0.30	0.30	0.29	0.29
	03	0.38	0.44	0.48	0.50	0.52	0.54	0.53	0.52
	04	0.55	0.63	0.70	0.74	0.77	0.79	0.80	0.78
	05	0.70	0.81	0.89	0.94	0.98	1.01	1.00	0.98
	06	0.86	0.99	1.09	1.15	1.20	1.24	1.23	1.21

1) The capacities for R507 deviate less than ± 2% from the capacities for R404A.

*Correction for subcooling  $\Delta t_{sub}$* 

When subcooling  $\Delta t_{sub}$  ahead of the expansion valve is other than 10°F, adjust the evaporator capacity by multiplying by the appropriate correction factor found in the following table.

Refrigerant	0°F	10°F	20°F	30°F	40°F	50°F	60°F	70°F	80°F	90°F
R404A/R507	1.18	1.00	0.89	0.81	0.74	0.68	0.63	0.60	0.56	0.53

*Metric conversions*

1 psi = 0.07 bar

 $\frac{5}{9} (t_1^\circ F - 32) = t_2^\circ C$ 

1 ton = 3.5 kW

**Design**

T 2 and TE 2 valves consist of four main components:

1. Thermostatic element
2. Orifice assembly
3. Valve body with connections
4. Superheat setting spindle

For a given valve type and refrigerant, the associated orifice is suitable for all versions of the valve body and in all evaporating temperature ranges.

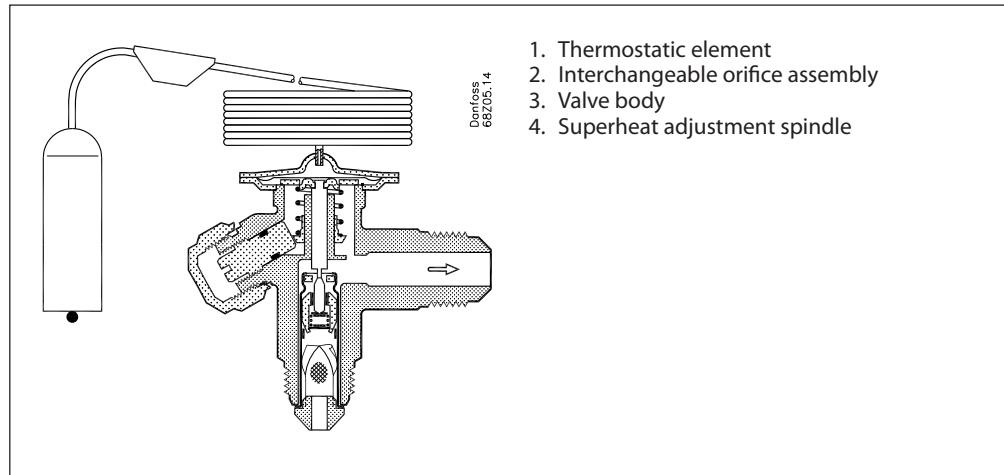
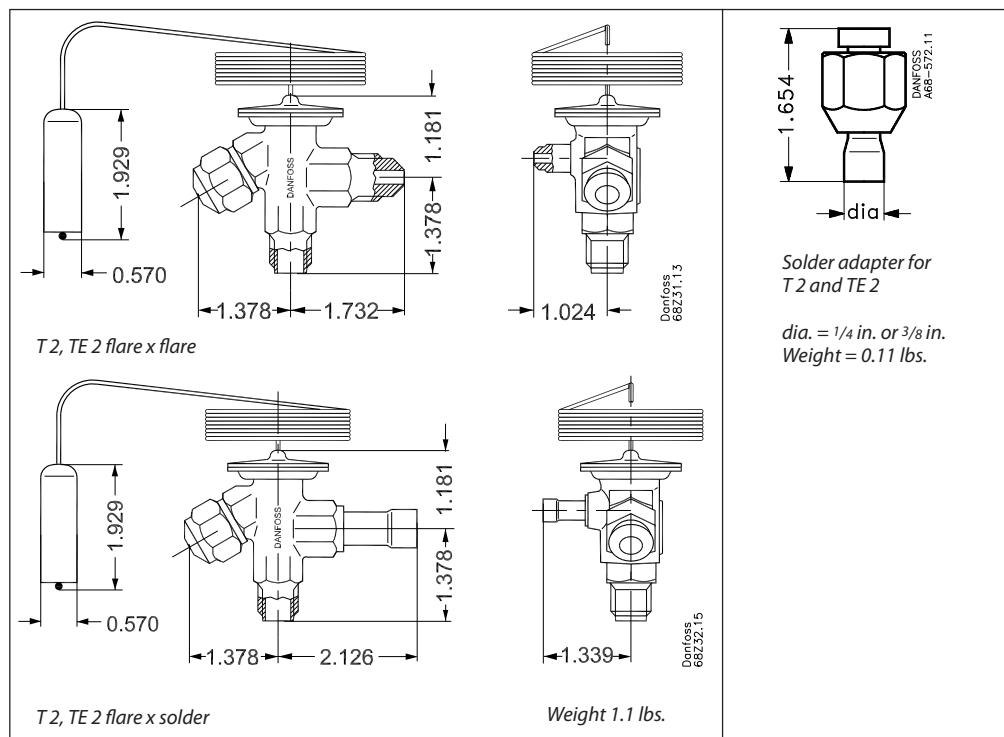
The valves are equipped with internal or external pressure equalization.

Where a liquid distributor is used, always use a valve with external equalization.

The double-contact bulb gives fast and precise reaction of temperature changes in the evaporator. The bulb design makes mounting quick and easy.

The valves are able to withstand the effects that normally occur with hot gas defrosting.

To insure long operating life, the valve cone and valve seat are made of a steel alloy with particularly good wear qualities.


**Dimensions and weights**


*Metric conversions*  
1 in. = 25.4 mm  
1 lb = 0.454 kg

**T 2/TE 2 operation and function**
**Superheat**

Superheat is the controlling parameter of the valve. Superheat, measured at the evaporator outlet, is defined as the difference between actual bulb temperature and the evaporating temperature at the saturation point. In other words, vapor is said to be superheated if its temperature is higher than the saturation temperature corresponding to its pressure. In order to prevent liquid refrigerant from entering the compressor, a certain minimum superheat must be maintained. Liquid entering the compressor causes serious damage. When discussing superheat, the following terms are used with respect to valve operation:

**Static Superheat**

Static superheat, SS, is the superheat above which the valve will begin to open.

**Opening Superheat**

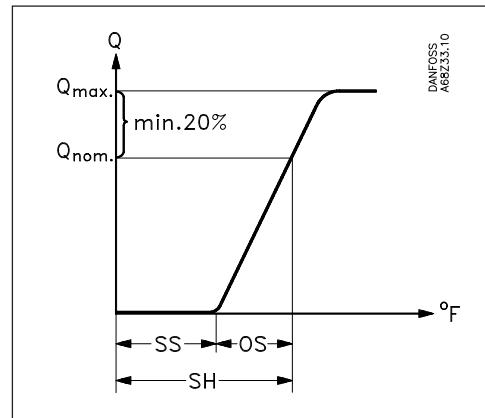
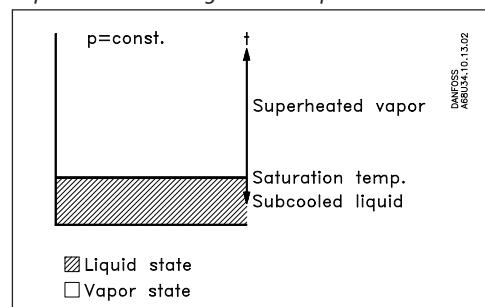
The opening superheat, OS, is the amount of superheat above the static superheat, SS, required to produce a given valve capacity.

**Subcooling  $\Delta t_{sub}$** 

Subcooling is defined as the difference between the liquid refrigerant temperature and its saturation temperature. Depending on system design, subcooling may be necessary to prevent flash gas from forming in the liquid line. If flash gas forms in the liquid line, the capacity of the expansion valve will be greatly reduced.

**Operating superheat**

The operating superheat, SH, is the sum of the static superheat, SS, and the opening superheat, OS.


**Superheat-subcooling relationship**

**MOP - Maximum Operating Pressure**

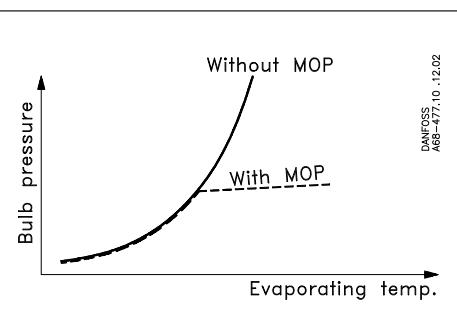
MOP function protects the compressor motor against overload during start-up.

MOP is the evaporating pressure at which the expansion valve will throttle liquid injection into the evaporator and thus prevent the evaporating pressure from rising.

MOP valves are primarily used in low temperature applications with a single evaporator and a single compressor. MOP occurs when the sensor reaches a predetermined maximum value.

Above MOP any increase in sensor temperature results in only minimal additional opening of the expansion valve.

An MOP charge is also known as a *pressure limiting charge*.


**Note!**

The MOP point will change if the factory superheat setting of the expansion valve is changed.

If the superheat setting increases, the MOP point is reduced and vice versa, if the MOP point increases, the superheat setting is reduced.

**Identification**

Main valve data is indicated on the valve element (fig. 1), and on the bottom of the orifice assembly (fig. 2).

*Example valve body*

TES2	= Type (E = external equalization)
068ZXXXX	= Code number
R404A - R507	= Refrigerant
<b>MOP 120 / +15°C</b>	= <b>MOP-point in psig / °C</b>
-40 / +10°C	= Evaporating temperature range in °C
<b>-40 / +50°F</b>	= Evaporating temperature range in °F
PB 34 bar/ <b>MWP 500 psig</b>	= Max. working pressure in bar/psig
4503A	= Date marking (week <b>45</b> , year 2003, weekday <b>A</b> = Monday)

*Orifice assembly for T 2 and TE 2 (Fig. 2)*

The orifice assembly is marked with the orifice size (06) and week stamp + last number in the year (174). The valve strainer is also shown in the figure.

The orifice assembly number is also given on the lid of its plastic container.

*Capillary tube label for T 2 and TE 2 (Fig. 3)*

The label gives the orifice size (04) and consists of the lid of the orifice assembly plastic container. It can easily be fastened around the expansion valve capillary tube to clearly identify the valve size.

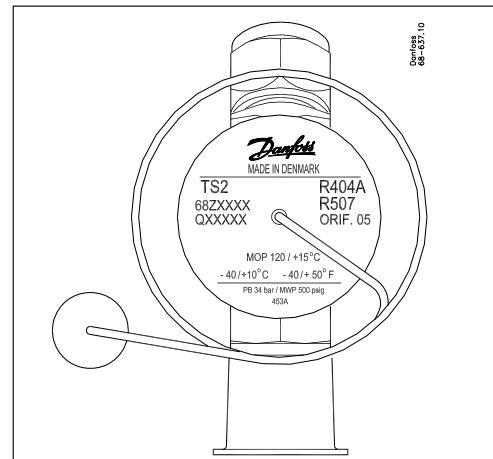


Fig. 1 Power element

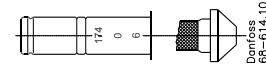


Fig. 2 Orifice assembly and filter for T 2 and TE 2

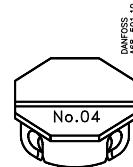


Fig. 3 Capillary tube label for T 2 and TE 2

*Metric conversions*

1 psi = 0.07 bar  
 $\frac{5}{9} (t_1^{\circ}\text{F} - 32) = t_2^{\circ}\text{C}$   
 1 ton = 3.5 kW  
 1 in. = 25.4 mm



---

Danfoss can accept no responsibility for possible errors in catalogues, brochures and other printed material. Danfoss reserves the right to alter its products without notice. This also applies to products already on order provided that such alterations can be made without subsequent changes being necessary in specifications already agreed.  
All trademarks in this material are property of the respective companies. Danfoss and the Danfoss logotype are trademarks of Danfoss A/S. All rights reserved.

---